A Root-Mean-Square-based Approach to Optimize a Parameter in the Control Systems Design

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Abstract—In the classical control systems design, a parameter is typically determined by the system's time response or frequency response. A standard unstable double-integrator plant with a spring constant K and a mass M is a perfect illustrative example. A stabilizing damper B is determined by setting the time-response to be qualitatively fast and non-oscillatory, which is basically setting the damping-ratio of the second order system equal to 1. The time-response is the system's response to a certain input, such as an impulse-function, a step-function, etc., not its response to an arbitrary disturbance, which is more likely to occur in the field The root-mean-square-(rms)-based approach introduced in this paper allows a designer to estimate a better parameter to minimize an objective function. The objective function is a combined quadratic function of the "error" and the "effort" rms values, both to be minimized by selecting the damper B accurately for each of the 4 (four) different schemes of disturbances. After successfully applied to the standard doubleintegrator plant given as an illustrative example, the similar approach is implemented for an armature-controlled dc motor speed-control design.

Keywords—root mean square; rms; objective function; doubleintegrator, disturbance, dc motor

I. INTRODUCTION

Classical methods to determine a parameter in the control systems design have been developed for almost a hundred years [1] since the steam engine's governor was invented. The most common methods are derived analytically from the system's responses to a certain input. The time response of a system to a step-input is one of the most common ways to characterize the performance of a control system [2]. The frequency response is also commonly used such as in methods based on the Nyquist criteria [3], or other methods.

Those methods based on the time response or the frequency response or both are usually valid and analytically verified, but in the field they - at least most of them - are not easily implemented due to a couple of reasons, among others for instance: (1) the methods require the use of sophisticated equipment such as an oscilloscope with the capability of displaying one-shot signals or a spectrum-analyzer for the frequency response, (2) the inputs should be a certain kind that is not easy to generate (an "ideal" step-function or an impulse-function $\delta(t)$ never actually exists), (3) even if the specific inputs may be approximated for some cases, they are

not always applicable, for example: a step input of armature voltage cannot be actually applied to a large DC motor.

Measuring equipments - particularly used in electrical engineering - mostly display the measurement results in rms values - also called "effective" values - especially when the signal being measured varies with time considerably [4]. Other physical units are usually converted into electrical quantities using some sensors, and then their rms values are measured.

II. AN ILLUSTRATIVE EXAMPLE

A well established case of control design in the classical control theory is an unstable double-integrator plant with a mass \mathbf{M} connected to a reference wall by a spring with the constant \mathbf{K} stabilized by a damper with the constant \mathbf{B} as represented by the block-diagram shown in Fig. 1.

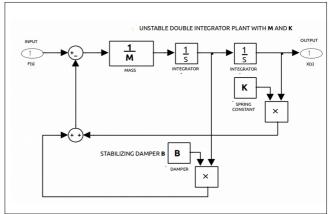


Fig. 1 An unstable double-integrator plant with a mass **M** and a spring constant **K**, stabilized by a damper B

The most common time-response is characterized by a step input. For a large ${\bf B}$, the response is over-damped, which is characterized as too slow, while for a small ${\bf B}$, the response is under-damped, characterized as too oscillatory. The "best" (both fast and non-oscillatory) performance is believed to be attained by setting the damping ratio $\xi=1$, or:

$$\mathbf{B} = 2\sqrt{\mathbf{M}\mathbf{K}} \tag{1}$$

The response for $\xi = 1$ is also characterized as critically damped response. A "normalized" plant with $\mathbf{M} = 1$ unit mass and $\mathbf{K} = 1$ unit spring constant requires $\mathbf{B} = \mathbf{2}$ to exhibit a critically damped response to a step input, as shown Fig. 2.

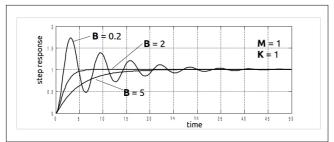


Fig. 2 The step-responses for $\mathbf{B} = 2$ (*critically damped*), for $\mathbf{B} = 5$ (*over-damped*) and for $\mathbf{B} = 0.2$ (*under-damped*)

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 Use either SI (MKS) or CGS as primary units. (SI units are encouraged.) English units may be used as secondary units (in parentheses). An exception would be the use of English units as identifiers in trade, such as "3.5-inch disk drive."

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$$a+b=\gamma$$
 (1) $\alpha+\beta=\chi$. (1) (1)

Note that the equation is centered using a center tab stop. Be sure that the symbols in your equation have been defined before or immediately following the equation. Use "(1)," not "Eq. (1)" or "equation (1)," except at the beginning of a sentence: "Equation (1) is ..."

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Table Head	Table Column Head		
	Table column subhead	Subhead	Subhead
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Sample of a Table footnote. (Table footnote)

Fig. 1. Example of a figure caption. (figure caption)

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